

IN THE SPECIFICATION

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The cable tensioner assembly 30 will now be described in further detail with respect to FIGS 2, 5, 6, and 7. FIG. 2 depicts a cross-sectional top view of the tool of FIG. 1 and FIG. 5 is a partial exploded view of the tool of FIG. 1 including the tool body, the cable tensioner assembly, and an exemplary hydraulically operated base tool. Fig 6 is a partial exploded view of the cable tensioner assembly 30 of FIG. 5 with some details omitted for clarity, and Fig. 7 depicts a cross sectional view of the cable tensioning assembly 30 of FIG. 5 taken along a rotation axis. The cable tensioning assembly 30 further includes a knob 80, a clutch tensioning force wheel 82 that includes an axle 90, a clutch ring 84, and a faceplate 86. The cable tensioner assembly 30 is rotatably mounted in the tool body 28 generally perpendicular to the elongate axis of the nose 32, so that the axle 90 of the wheel 82 extends substantially through an opening 70 in the body 28, allowing access of the axle end 102 at the opposite side of the body. In an embodiment, the axle 90 rides in a ring bearing 106 mounted within the body 28 and sandwiched between two bearing washers 108. In addition, a circumferential groove is formed in the axle near the end 102 to allow fastening of a retaining ring 104 to rotatably retain the axle 90 in the body 28.

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On the body side 110 of the wheel 82, the surface of the wheel 82 has a flat portion 113 extending radially away from the axle 90, then the surface assumes a sloping profile 444 extending from the flat portion 113 to a rim 81 of the wheel 82 forming a flared portion 114. The flat portion 113 is configured to allow attaching a faceplate 86 thereto, with, for example, screws 136. An attachment portion 88 of the faceplate 86 is a raised cylindrical platform having a height perpendicular to a face 87 of the faceplate 86. In an aspect of the invention, the height may be slightly smaller than the diameter of the safety cable 12. When the faceplate 86 is attached to the wheel 82, the face 87 of the faceplate 86 and the ~~sloping profile~~ flared portion 114 forms a gripping slot 115

tapering to a relatively smaller width toward the axle 90. Accordingly, the ~~sloping~~ profile flared portion 114 acts to wedge a safety cable 12 wrapped circumferentially in the gripping slot 115, thereby retaining the cable 12 so that tension can be applied by rotating the knob 80.

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The clutch ring 84 fits rotatably around the rim 81 of the wheel 82 and, as more clearly shown in Fig. 5A 6, includes indentations 120 laterally formed and uniformly spaced in the inner diameter of the ring 84. The indentations 120 movably accommodate the ends 121 of the clutch tensioning force pins 116 radially protruding from rim 81 of the wheel 82. For example, the indentations may have a circular cross section sized to engage a hemispherically shaped end 121 of the clutch tensioning force pin 116. The ends 121 of the clutch tensioning force pins 116, forced into the indentations 120 by action of the spring actuator 98 and the compression spring 96, prevent rotation of the clutch ring 84 around the wheel 82 until a rotational force is applied sufficient to overcome the force of the compression spring 96 communicated through the spring actuator 98 to the clutch tensioning force pins 116 lodged in respective indentations 120. When sufficient rotational force, or tension, is applied to the clutch ring 84, the sides of the indentations 120 act to radially displace the protruding clutch tensioning force pins 116 in a direction towards the axle bore 92 and out of the indentations 120, so that the clutch ring 84 rotates about the wheel 82 as long as sufficient rotational force is applied. The force acting on the clutch tensioning force pins 116 to keep the pins 116 lodged in the indentations 120, and, correspondingly, the rotational force required to overcome the radial force on the pins 116, can be adjusted by threading the adjustment screw 100 in or out of the axle bore 92 to alter the compression of the spring 96, accessed via an axle bore 92 opening at the axle end 102 as shown Fig. 5. For example, as the adjustment screw 100 is tightened, increasing pressure is exerted on the spring actuator 98 corresponding to the compression on the spring. The force on the spring actuator 98 is transferred longitudinally to the tensioning force pins 116.